

State Space Models, Linearization, Transfer Function

Automatic Control, Basic Course, Lecture 2

October 29, 2019

Lund University, Department of Automatic Control

1. State Space Models
2. Linearization
3. Transfer Function
4. Block Diagram Representation

- PID-control
- State-space model of plant

State Space Models

State Space Models

Consider a linear differential equation of order n

$$\frac{d^n y}{dt^n} + a_1 \frac{d^{n-1} y}{dt^{n-1}} + \dots + a_n y = b_0 \frac{d^n u}{dt^n} + b_1 \frac{d^{n-1} u}{dt^{n-1}} + \dots + b_n u$$

For linear systems **the superposition principle** holds:

$$u = u_1 \implies y = y_1 \text{ and}$$

$$u = u_2 \implies y = y_2 \text{ implies}$$

$$u = c_1 \cdot u_1 + c_2 \cdot u_2 \implies y = c_1 \cdot y_1 + c_2 \cdot y_2$$

and vice versa; We can consider the output from a sum of signals by considering the influence from each component.

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Q: Why is this not true for nonlinear systems? Example?

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An **alternative** to ONE differential equation of order n^{th} is to write it as a system of **n coupled differential equations, each of order one.**

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General State space representation:

$$\left\{ \begin{array}{l} \dot{x}_1 = f_1(x_1, x_2, \dots, x_n, u) \\ \dot{x}_2 = f_2(x_1, x_2, \dots, x_n, u) \\ \dots \\ \dot{x}_n = f_n(x_1, x_2, \dots, x_n, u) \\ y = g(x_1, x_2, \dots, x_n, u) \end{array} \right.$$

The last row is a static equation relating the introduced **states** (x) with the input u , and the output y .

State Space Models

Consider a linear differential equation of order n

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An **alternative** to ONE differential equation of order n^{th} is to write it as a system of n coupled differential equations, each of order one.

Linear state space representation:

$$\begin{cases} \dot{X}_1 = a_{11}X_1 + \dots + a_{1n}X_n + b_1 u \\ \dot{X}_2 = a_{21}X_1 + \dots + a_{2n}X_n + b_2 u \\ \dots \\ \dot{X}_n = a_{n1}X_1 + \dots + a_{nn}X_n + b_n u \\ y = c_1 X_1 + c_2 X_2 + \dots + c_n X_n + du \end{cases} \quad \begin{cases} \begin{bmatrix} \dot{X}_1 \\ \dot{X}_2 \\ \vdots \\ \dot{X}_n \end{bmatrix} = \begin{bmatrix} a_{11} & a_{12} & \dots & a_{1n} \\ a_{21} & a_{22} & \dots & a_{2n} \\ \vdots & \vdots & \ddots & \vdots \\ a_{n1} & a_{n2} & \dots & a_{nn} \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ \vdots \\ X_n \end{bmatrix} + \begin{bmatrix} b_1 \\ b_2 \\ \vdots \\ b_n \end{bmatrix} u \\ \\ y = [c_1 \quad c_2 \quad \dots \quad c_n] \begin{bmatrix} X_1 \\ X_2 \\ \vdots \\ X_n \end{bmatrix} + du \end{cases}$$

State Space Models

Consider a linear differential equation of order n

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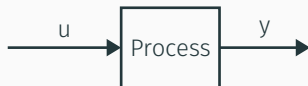
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Linear state space representation:

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NOTE: **Only states (x) and inputs (u) are allowed** on the right hand side in Eq.-system above (in f and g) for it to be called a state-space representation!

State Space Models



Linear dynamics can be described in the following form

$$\dot{x} = Ax + Bu$$

$$y = Cx (+Du)$$

Here $x \in \mathbb{R}^n$ is a vector with states. States can have a physical "interpretation", but not necessary.

In this course $u \in \mathbb{R}$ and $y \in \mathbb{R}$ will be scalars.

(For MIMO systems, see Multivariable Control (FRTN10))

Example

Example

The position of a mass m controlled by a force u is described by

$$m\ddot{x} = u$$

where x is the position of the mass.



Introduce the states $x_1 = \dot{x}$ and $x_2 = x$ and write the system on state space form. Let the position be the output.

Dynamical Systems

	Continuous Time	Discrete Time (sampled)
Linear	This course	Real-Time Systems / Signal proc. (FRTN01)
Nonlinear	Nonlinear Control and Servo Systems (FRTN05)	

Linearization

Linearization - Why?

Many systems are nonlinear. However, one can approximate them with linear ones. This to get a system that is easier to analyze.

A few examples of nonlinear systems:

- Water tanks (Labs 1,2)
- Air resistance
- Action potentials in neurons
- Pendulum under the influence of gravity
- ...

Linearization - How?

Given a nonlinear system $\dot{x} = f(x, u)$, $y = g(x, u)$

Linearization - How?

Given a nonlinear system $\dot{x} = f(x, u)$, $y = g(x, u)$

1. Determine a **stationary point** (x_0, u_0) to linearize around

$$\dot{x}_0 = 0 \quad \Leftrightarrow \quad f(x_0, u_0) = 0$$

Linearization - How?

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1. Determine a **stationary point** (x_0, u_0) to linearize around

$$\dot{x}_0 = 0 \quad \Leftrightarrow \quad f(x_0, u_0) = 0$$

2. Make a first order **Taylor series expansions** of f and g around (x_0, u_0) :

$$f(x, u) \approx f(x_0, u_0) + \frac{\partial}{\partial x} f(x_0, u_0)(x - x_0) + \frac{\partial}{\partial u} f(x_0, u_0)(u - u_0)$$

$$g(x, u) \approx g(x_0, u_0) + \frac{\partial}{\partial x} g(x_0, u_0)(x - x_0) + \frac{\partial}{\partial u} g(x_0, u_0)(u - u_0)$$

Notice that $f(x_0, u_0) = 0$ and let $y_0 = g(x_0, u_0)$

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Notice that $f(x_0, u_0) = 0$ and let $y_0 = g(x_0, u_0)$

3. **Introduce** $\Delta x = x - x_0$, $\Delta u = u - u_0$ and $\Delta y = y - y_0$

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Notice that $f(x_0, u_0) = 0$ and let $y_0 = g(x_0, u_0)$

3. **Introduce** $\Delta x = x - x_0$, $\Delta u = u - u_0$ and $\Delta y = y - y_0$
4. The state-space equations in the new variables are given by:

$$\dot{\Delta x} = \dot{x} - \dot{x}_0 = f(x, u) \approx \frac{\partial}{\partial x} f(x_0, u_0)\Delta x + \frac{\partial}{\partial u} f(x_0, u_0)\Delta u = A\Delta x + B\Delta u$$

$$\Delta y = g(x, u) - y_0 \approx \frac{\partial}{\partial x} g(x_0, u_0)\Delta x + \frac{\partial}{\partial u} g(x_0, u_0)\Delta u = C\Delta x + D\Delta u$$

Example - Linearization

Example

The dynamics of a specific system is described by

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = -\frac{x_2^4}{x_1^2} + x_1 + \sqrt{u+1}$$

$$y = x_1^2 + u^2$$

- a) Find all stationary points
- b) Linearize the system around the stationary point corresponding to $u_0 = 3$

The dynamics of a specific system is described by

$$\begin{aligned}\dot{x}_1 &= x_2 &&= f_1(x_1, x_2, u) \\ \dot{x}_2 &= -\frac{x_2^4}{x_1^2} + x_1 + \sqrt{u+1} &&= f_2(x_1, x_2, u) \\ y &= x_1^2 + u^2 &&= g(x_1, x_2, u)\end{aligned}$$

(a) Find stationary point for $u_0 = 3$: ($\dot{x}_1 = \dot{x}_2 = 0$)

$$0 = x_2$$

$$0 = -\frac{x_2^4}{x_1^2} + x_1 + \sqrt{3+1}$$

$$y = x_1^2 + 3^2$$

$$\implies (x_{10}, x_{20}, u_0) = (-2, 0, 3)$$

$$y_0 = g(x_{10}, x_{20}, u_0) = 13$$

The dynamics of a specific system is described by

$$\dot{x}_1 = x_2 = f_1(x_1, x_2, u)$$

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(b) Linearize around stationary point $(-2, 0, 3)$

$$\frac{\partial f_1}{\partial x_1} = 0,$$

$$\frac{\partial f_1}{\partial x_2} = 1,$$

$$\frac{\partial f_1}{\partial u} = 0,$$

$$\frac{\partial f_2}{\partial x_1} = +2\frac{x_2^4}{x_1^3} + 1,$$

$$\frac{\partial f_2}{\partial x_2} = -4\frac{x_2^3}{x_1^2},$$

$$\frac{\partial f_2}{\partial u} = \frac{1}{2\sqrt{u+1}},$$

$$\frac{\partial g}{\partial x_1} = 2x_1,$$

$$\frac{\partial g}{\partial x_2} = 0,$$

$$\frac{\partial g}{\partial u} = 2u,$$

The dynamics of a specific system is described by

$$\dot{x}_1 = x_2 = f_1(x_1, x_2, u)$$

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$$y = x_1^2 + u^2 = g(x_1, x_2, u)$$

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(b) Linearize around stationary point $(-2, 0, 3)$

$$\frac{\partial f_1}{\partial x_1} \Big|_{\{x_0, u_0\}} = 0, \quad \frac{\partial f_1}{\partial x_2} \Big|_{\{x_0, u_0\}} = 1, \quad \frac{\partial f_1}{\partial u} \Big|_{\{x_0, u_0\}} = 0,$$

$$\frac{\partial f_2}{\partial x_1} \Big|_{\{x_0, u_0\}} = 1, \quad \frac{\partial f_2}{\partial x_2} \Big|_{\{x_0, u_0\}} = 0, \quad \frac{\partial f_2}{\partial u} \Big|_{\{x_0, u_0\}} = \frac{1}{4},$$

$$\frac{\partial g}{\partial x_1} \Big|_{\{x_0, u_0\}} = -4, \quad \frac{\partial g}{\partial x_2} \Big|_{\{x_0, u_0\}} = 0, \quad \frac{\partial g}{\partial u} \Big|_{\{x_0, u_0\}} = 6,$$

The dynamics of a specific system is described by

$$\dot{x}_1 = x_2 = f_1(x_1, x_2, u)$$

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$$y = x_1^2 + u^2 = g(x_1, x_2, u)$$

$$\implies (x_{10}, x_{20}, u_0) = (-2, 0, 3)$$

$$y_0 = g(x_{10}, x_{20}, u_0) = 13$$

(b) Linearize around stationary point $(-2, 0, 3)$

$$\frac{\partial f(x, u)}{\partial x} \Big|_{\{x_0, u_0\}} = A = \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix} \quad \frac{\partial f(x, u)}{\partial u} \Big|_{\{x_0, u_0\}} = B = \begin{bmatrix} 0 \\ \frac{1}{4} \end{bmatrix}$$

$$\frac{\partial g(x, u)}{\partial x} \Big|_{\{x_0, u_0\}} = C = \begin{bmatrix} -4 & 0 \end{bmatrix} \quad \frac{\partial g(x, u)}{\partial u} \Big|_{\{x_0, u_0\}} = D = \begin{bmatrix} 6 \end{bmatrix}$$

The dynamics of a specific system is described by

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$$y = x_1^2 + u^2 = g(x_1, x_2, u)$$

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$$y_0 = g(x_{10}, x_{20}, u_0) = 13$$

Introduce

$$\Delta x_1 = x_1 - x_{10}, \quad \Delta x_2 = x_2 - x_{20}$$

$$\Delta u = u - u_0 \quad \Delta y = y - y_0$$

The state-space equations in the new variables are given by:

$$\begin{bmatrix} \frac{\Delta x_1}{dt} \\ \frac{\Delta x_2}{dt} \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix} \begin{bmatrix} \Delta x_1 \\ \Delta x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{4} \end{bmatrix} u$$

$$\Delta y = \begin{bmatrix} -4 & 0 \end{bmatrix} \begin{bmatrix} \Delta x_1 \\ \Delta x_2 \end{bmatrix} + \begin{bmatrix} 6 \end{bmatrix} u$$

Transfer Function

Laplace Transformation

Let $f(t)$ be a function of time t , the Laplace transformation $\mathcal{L}(f(t))(s)$ is defined as

$$\mathcal{L}(f(t))(s) = F(s) = \int_0^{\infty} e^{-st}f(t)dt$$

Example:

$$\mathcal{L}\left(\frac{df(t)}{dt}\right)(s) = sF(s) - f(0)$$

Initial values helps to calculate what happens in transient phase!

Assuming that $f(0) = f'(0) = \dots = f^{n-1}(0) = 0$ (common assumption during this course, but not always!!) it has the property that

$$\mathcal{L}\left(\frac{d^n f(t)}{dt^n}\right)(s) = s^n F(s)$$

$$\mathcal{L}\left(\int_0^t f(\tau) \frac{de}{d\tau}\right)(s) = \frac{1}{s} F(s) \quad (\text{integrator})$$

See Collection of Formulae for a table of Laplace transformations.

Example - Transfer Function

Example

A system's dynamics is described by the differential equation

$$\ddot{y} + a_1\dot{y} + a_2y = b_1\dot{u} + b_2u.$$

After Laplace transformation we get

$$(s^2 + a_1s + a_2)Y(s) = (b_1s + b_2)U(s)$$

which can be written as

$$Y(s) = \frac{\overbrace{b_1s + b_2}^{G(s)}}{s^2 + a_1s + a_2} U(s) = G(s)U(s)$$

$G(s)$ is called the transfer function of the system.

Transfer Function

Relation between control signal $U(s)$ and output $Y(s)$:

$$Y(s) = G(s)U(s)$$

$G(s)$ often fraction of polynomial, i.e.,

$$G(s) = \frac{Q(s)}{P(s)}$$

Zeros of $Q(s)$ are called zeros of the system, zeros of $P(s)$ are called poles of the system.

The poles play a very important role for the system's behavior.

From State Space to Transfer Function

For a system on state space form

$$\dot{x} = Ax + Bu$$

$$y = Cx + Du$$

the transfer function is given by

$$G(s) = C(sI - A)^{-1}B + D$$

Observe: the denominator of $G(s)$ is given by $P(s) = \det(sI - A)$, so eigenvalues of A are poles of the system.

From Transfer Function to State Space

Can be done in several ways, see Collection of Formulae.

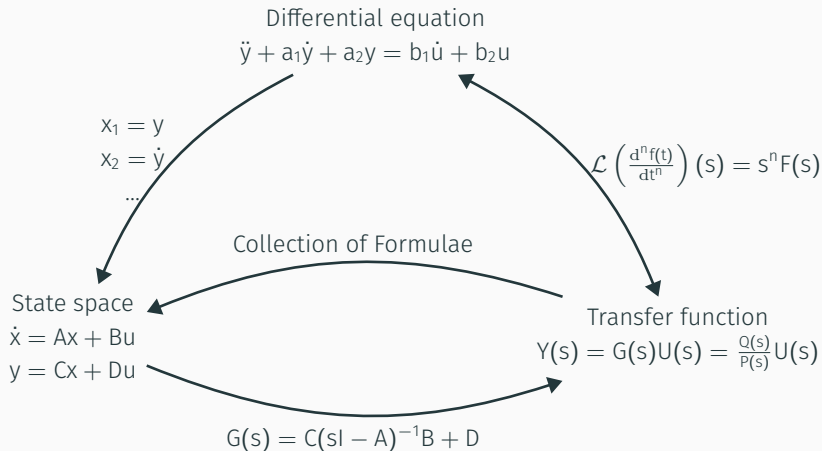
Example

A system's transfer function is

$$G(s) = \frac{2s + 1}{s^3 + 4s - 8}$$

Write the system on a state space form of your choice.

Three Ways to Describe a Dynamical System



Block Diagram Representation

Block Diagram - Transfer Function

When the blocks in a block diagram are replaced by transfer functions, it is possible to describe the relations between signals in an easy way.

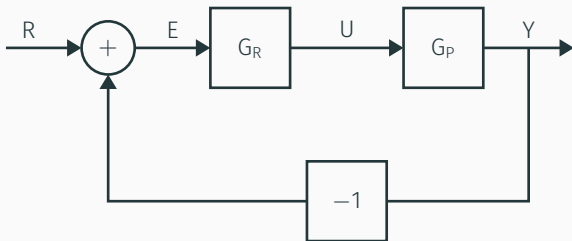


$$Y(s) = G_P(s)U(s)$$

Block Diagram - Components

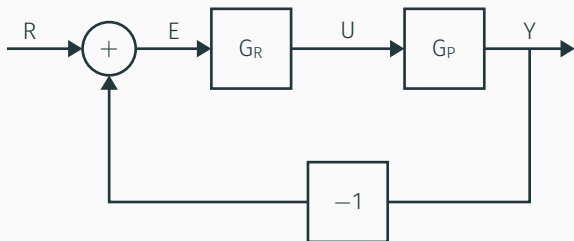
Most block diagrams consist of three components:

- Blocks - Transfer functions
- Arrows - Signals
- Summations



where R , E , U , Y are the Laplace transformations of the reference $r(t)$, control error $e(t)$, control signal $u(t)$, and output $y(t)$, respectively.

Determine Transfer Function From Block Diagram



$$Y = G_P U, \quad U = G_R E, \quad E = R - Y$$

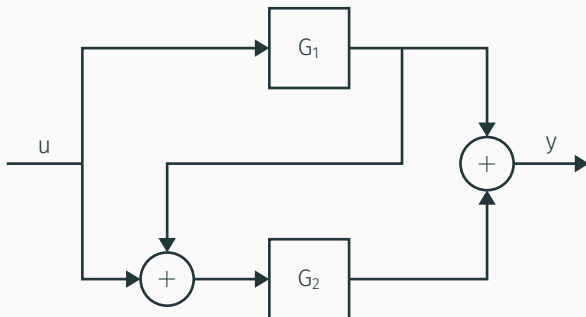
From the equations above the transfer function between r and y is

$$Y = \frac{G_P G_R}{1 + G_P G_R} R$$

Example - Transfer Functions

Example

Two systems, G_1 and G_2 , are interconnected as in the figure below



Compute the transfer function from u to y , G_{yu} .

Summary

This lecture

1. State Space Models
2. Linearization
3. Transfer Function
4. Block Diagram Representation

Next lecture

- Impulse Response Analysis
- Step Response Analysis