



LUNDS
UNIVERSITET

Lecture 2

FRTN10 Multivariable Control

Automatic Control LTH, 2019





Course Outline

- L1–L5 Specifications, models and loop-shaping by hand
 - 1 Introduction
 - 2 **Stability and robustness**
 - 3 Specifications and disturbance models
 - 4 Control synthesis in frequency domain
 - 5 Case study: DVD player
- L6–L8 Limitations on achievable performance
- L9–L11 Controller optimization: analytic approach
- L12–L14 Controller optimization: numerical approach
- L15 Course review



L2: Stability and robustness

- 1 Stability
- 2 Sensitivity and robustness
- 3 The Small Gain Theorem
- 4 Singular values



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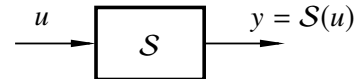
Stability is crucial

Examples:

- bicycle
- JAS 39 Gripen
- Mercedes A-class
- ABS brakes



Input-output stability



A general system \mathcal{S} is called **input-output stable** (or “ L_2 stable” or “BIBO stable” or just “stable”) if its L_2 gain is finite:

$$\|\mathcal{S}\| = \sup_u \frac{\|\mathcal{S}(u)\|}{\|u\|} < \infty$$



Input-output stability of LTI systems

For an LTI system \mathcal{S} with impulse response $g(t)$ and transfer function $G(s)$, the following stability conditions are equivalent:

- $\|\mathcal{S}\|$ is bounded
- $g(t)$ decays exponentially
- All poles of $G(s)$ are in the left half-plane (LHP), i.e., all poles have negative real part



Internal stability

The LTI system

$$\frac{dx}{dt} = Ax + Bu$$
$$y = Cx + Du$$

is called **internally stable** if the following equivalent conditions hold:

- The state x decays exponentially when $u = 0$
- All eigenvalues of A are in the LHP



Internal vs input-output stability

If

$$\dot{x} = Ax + Bu$$

$$y = Cx + Du$$

is internally stable **then**

$$G(s) = C(sI - A)^{-1}B + D$$

is input-output stable.

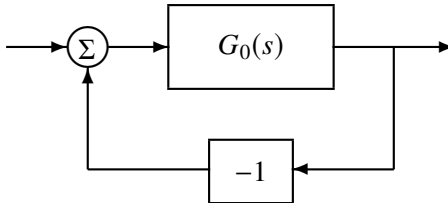
Warning

The opposite is not always true! There may be unstable pole-zero cancellations (that also render the system uncontrollable and/or unobservable), and these may not be seen in the transfer function!



Stability of feedback loops

Assume scalar open-loop system $G_0(s)$



The closed-loop system is stable **if and only if** all solutions to the characteristic equation

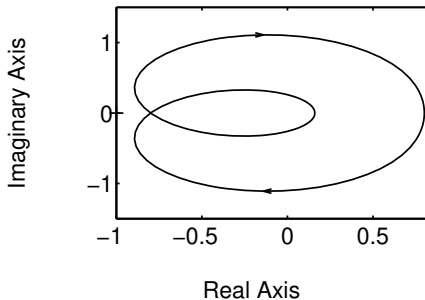
$$1 + G_0(s) = 0$$

are in the left half-plane.



Simplified Nyquist criterion

If $G_0(s)$ is stable, then the closed-loop system $[1 + G_0(s)]^{-1}$ is stable **if and only if** the Nyquist curve of $G_0(s)$ does not encircle -1 .



(Note: Matlab gives a Nyquist plot for both positive and negative frequencies)



General Nyquist criterion

Let

- P = number of **unstable** (RHP) poles in $G_0(s)$
- N = number of **clockwise** encirclements of -1 by the Nyquist plot of $G_0(s)$

Then the closed-loop system $[1 + G_0(s)]^{-1}$ has $P + N$ unstable poles



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Sensitivity and robustness

- How sensitive is the closed-loop system to model errors and disturbances?
- How do we measure the “distance to instability”?
- Is it possible to guarantee stability for all systems within some distance from the ideal model?



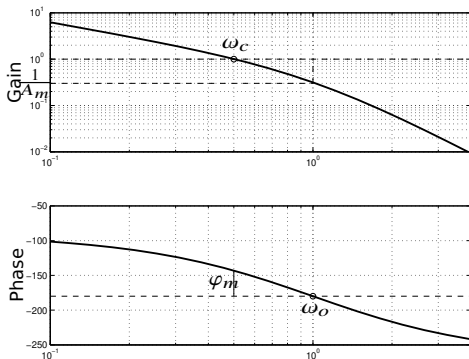
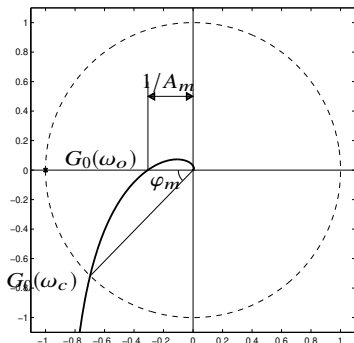
Amplitude and phase margins

Amplitude margin A_m :

$$\arg G_0(i\omega_0) = -180^\circ, \quad |G_0(i\omega_0)| = 1/A_m$$

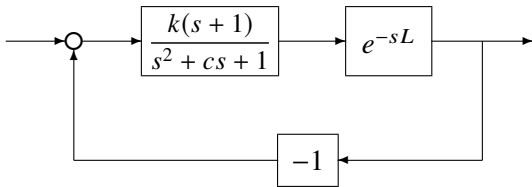
Phase margin φ_m :

$$|G_0(i\omega_c)| = 1, \quad \arg G_0(i\omega_c) = \varphi_m - 180^\circ$$

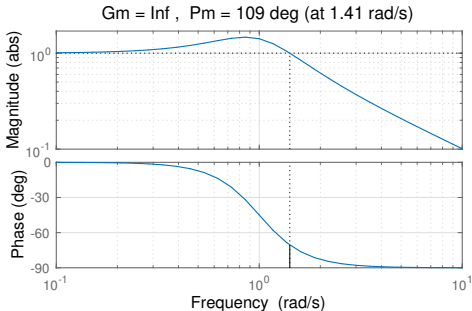
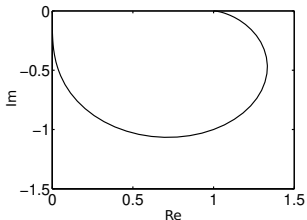




Mini-problem



Nominally $k = 1$, $c = 1$ and $L = 0$. How much margin is there in each parameter before the closed-loop system becomes unstable?

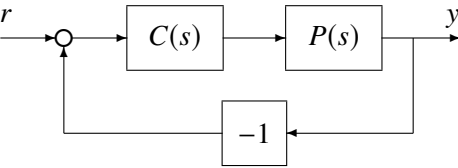




Mini-problem



Sensitivity functions



$$S(s) = \frac{1}{1 + P(s)C(s)} \quad \text{sensitivity function}$$

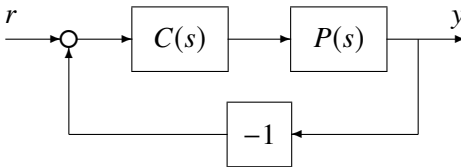
$$T(s) = \frac{P(s)C(s)}{1 + P(s)C(s)} \quad \text{complementary sensitivity function}$$

Note that we always have

$$S(s) + T(s) = 1$$



Sensitivity towards changes in plant



How sensitive is the closed loop to a (small) change in P ?

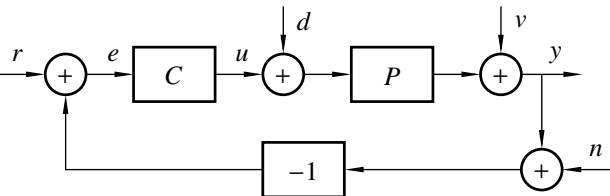
$$\frac{dT}{dP} = \frac{C}{(1 + PC)^2} = \frac{T}{P(1 + PC)}$$

Relative change in T compared to relative change in P :

$$\frac{dT/T}{dP/P} = \frac{1}{1 + PC} = S$$



Sensitivity towards disturbances



Open-loop response ($C = 0$) to process disturbances d, v :

$$Y_{ol} = V + PD$$

Closed-loop response:

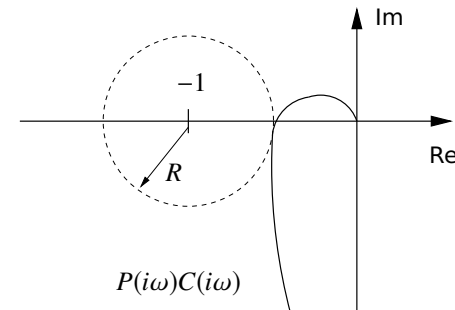
$$Y_{cl} = \frac{1}{1 + PC}V + \frac{P}{1 + PC}D = SY_{ol}$$



Interpretation as stability margin

The maximum gain of the sensitivity function measures the inverse of the distance between the Nyquist plot and the point -1 :

$$R^{-1} = \sup_{\omega} \left| \frac{1}{1 + P(i\omega)C(i\omega)} \right| = M_s$$





L2: Stability and robustness

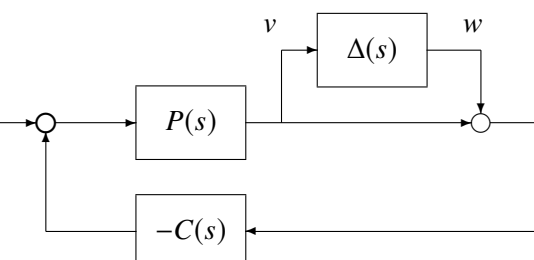
- 1 Stability
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Robustness analysis

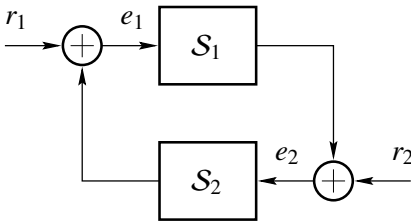
How large plant uncertainty Δ can be tolerated without risking instability?

Example (multiplicative uncertainty):





The Small Gain Theorem

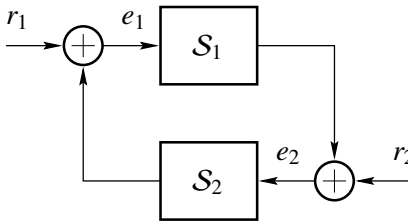


Assume that \mathcal{S}_1 and \mathcal{S}_2 are stable. **If** $\|\mathcal{S}_1\| \cdot \|\mathcal{S}_2\| < 1$, **then** the closed-loop system (from (r_1, r_2) to (e_1, e_2)) is stable.

- Note 1: The theorem applies also to nonlinear, time-varying, and multivariable systems
- Note 2: The stability condition is sufficient but not necessary, so the results may be conservative



The Small Gain Theorem



Assume that \mathcal{S}_1 and \mathcal{S}_2 are stable. **If** $\|\mathcal{S}_1\| \cdot \|\mathcal{S}_2\| < 1$, **then** the closed-loop system (from (r_1, r_2) to (e_1, e_2)) is stable.

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Proof

$$e_1 = r_1 + \mathcal{S}_2(r_2 + \mathcal{S}_1(e_1))$$

$$\|e_1\| \leq \|r_1\| + \|\mathcal{S}_2\|(\|r_2\| + \|\mathcal{S}_1\| \cdot \|e_1\|)$$

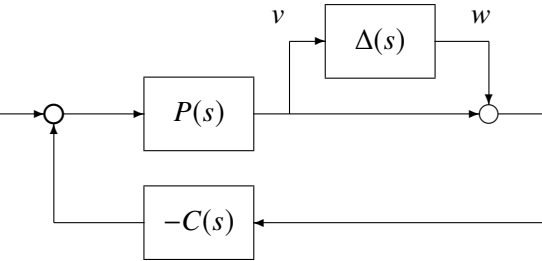
$$\|e_1\| \leq \frac{\|r_1\| + \|\mathcal{S}_2\| \cdot \|r_2\|}{1 - \|\mathcal{S}_1\| \cdot \|\mathcal{S}_2\|}$$

This shows bounded gain from (r_1, r_2) to e_1 .

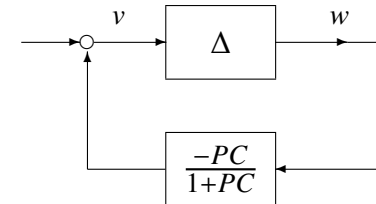
The gain to e_2 is bounded in the same way.



Application to robustness analysis

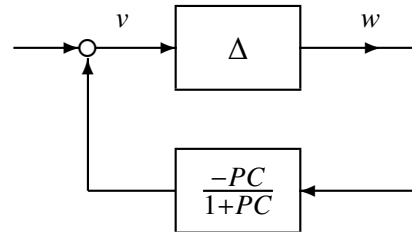


The diagram can be redrawn as





Application to robustness analysis



Assuming that $T = \frac{PC}{1+PC}$ is stable, the Small Gain Theorem guarantees stability if

$$\|\Delta\| \cdot \|T\| < 1$$



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Gain of multivariable systems

Recall from Lecture 1 that

$$\|\mathcal{S}\| = \sup_{\omega} |G(i\omega)| = \|G\|_{\infty}$$

for a stable LTI system \mathcal{S} .

How to calculate $|G(i\omega)|$ for a multivariable system?



Vector norm and matrix gain

For a vector $x \in \mathbf{C}^n$, we use the 2-norm

$$|x| = \sqrt{x^*x} = \sqrt{|x_1|^2 + \dots + |x_n|^2}$$

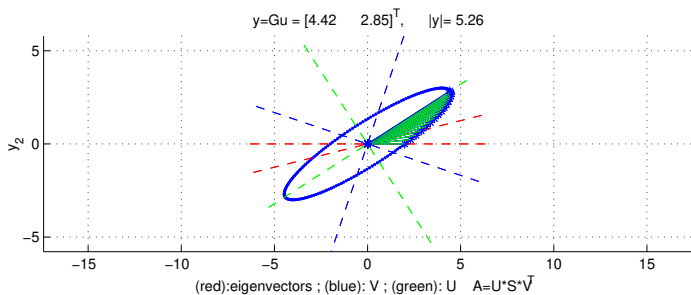
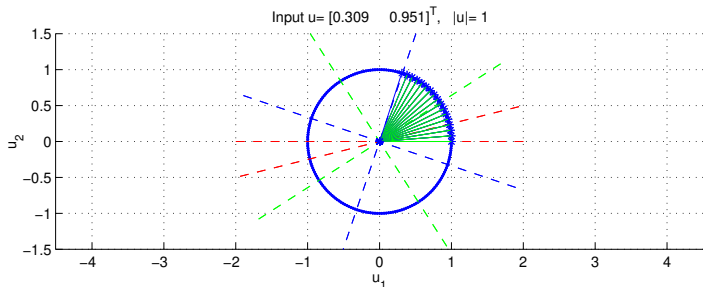
(A^* denotes the conjugate transpose of A)

For a matrix $A \in \mathbf{C}^{n \times m}$, we use the L_2 -induced norm

$$\|A\| := \sup_x \frac{|Ax|}{|x|} = \sup_x \sqrt{\frac{x^* A^* A x}{x^* x}} = \sqrt{\bar{\lambda}(A^* A)}$$

$\bar{\lambda}(A^* A)$ denotes the largest eigenvalue of $A^* A$. The ratio $|Ax|/|x|$ is maximized when x is a corresponding eigenvector.

Example: Different gains in different directions: $\begin{bmatrix} y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} 2 & 4 \\ 0 & 3 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$





Singular Values

For a matrix A , its singular values σ_i are defined as

$$\sigma_i = \sqrt{\lambda_i}$$

where λ_i are the eigenvalues of A^*A .

Let $\overline{\sigma}(A)$ denote the largest singular value and $\underline{\sigma}(A)$ the smallest singular value.

For a linear map $y = Ax$, it holds that

$$\underline{\sigma}(A) \leq \frac{|y|}{|x|} \leq \overline{\sigma}(A)$$

The singular values are typically computed using singular value decomposition (SVD)



Singular value decomposition (SVD)

Let A be an $m \times n$ complex matrix. It can be factored as

$$A = U\Sigma V^*$$

where

- U is an $m \times m$ unitary complex matrix, whose columns represent different **output directions**
- Σ is an $m \times n$ matrix with non-negative real numbers (the singular values) on the diagonal, representing different **gains**
- V is an $n \times n$ unitary complex matrix, whose columns represent different **input directions**

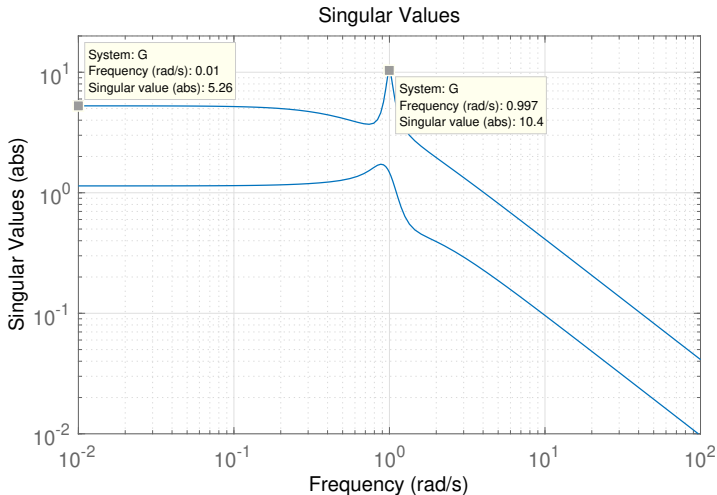
With $A = G(i\omega)$, the complex directions reveal both the relative magnitude and phase of the input/output signals with frequency ω .



Example: Gain of multivariable system

$$G(s) = \begin{bmatrix} \frac{2}{s+1} & \frac{4}{2s+1} \\ \frac{s}{s^2+0.1s+1} & \frac{3}{s+1} \end{bmatrix}$$

```
>> s = zpk('s');  
>> G = [2/(s+1) 4/(2*s+1); s/(s^2+0.1*s+1) 3/(s+1)];  
>> sigma(G) % plot sigma values of G wrt freq  
>> [gain,w] = norm(G,inf) % infinity norm = system gain  
gain =  
    10.3698  
w =  
    1.0000
```



The singular values of the transfer function matrix (prev slide). Note that $G(0) = \begin{bmatrix} 2 & 4 \\ 0 & 3 \end{bmatrix}$ (prev example). $\|G\|_{\infty} = \|G(1i)\| = 10.3698$.

```

>> A = evalfr(G,i*1)
A =
    1.0000 - 1.0000i    0.8000 - 1.6000i
   10.0000 - 0.0000i    1.5000 - 1.5000i

>> [U,S,V] = svd(A)
U =
   -0.1307 + 0.1082i    0.9472 - 0.2720i
   -0.9855 + 0.0023i   -0.1557 - 0.0675i
S =
   10.3698             0
         0         1.4720
V =
   -0.9734 + 0.0000i   -0.2292 + 0.0000i
   -0.1697 - 0.1541i    0.7206 + 0.6544i

```



Summary of Lecture 2

- Input-output stability: $\|\mathcal{S}\| < \infty$
- Sensitivity function: $S(s) := \frac{1}{1+P(s)C(s)}$
 - Three different interpretations
- Small Gain Theorem: The feedback interconnection of \mathcal{S}_1 and \mathcal{S}_2 is stable **if** $\|\mathcal{S}_1\| \cdot \|\mathcal{S}_2\| < 1$
 - Conservative compared to the Nyquist criterion
 - Useful for robustness analysis
- The gain of a multivariable system $G(s)$ is given by $\sup_{\omega} \bar{\sigma}(G(i\omega))$, where $\bar{\sigma}$ is the largest singular value
 - Singular values by SVD (on computer): $G(i\omega) = U\Sigma V^*$