Transient Response, Step Response Analysis

Automatic Control, Basic Course, Lecture 3

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1. Transient Response

2. Step Response Analysis

Transient Response

Given a system on state space form

$$\dot{x} = Ax + Bu$$
$$y = Cx + Du$$

The solution,
$$y(t)$$
, is then given by

$$y(t) = \frac{Ce^{At}x(0)}{Ce^{At}x(0)} + C\int_{0}^{t} e^{A(t-\tau)}Bu(\tau)d\tau + Du(t)$$

Initial state, uninteresting except when the controller is initialized

Weighted integral of the control signal, interesting part Direct term, often neglectable in practical systems Shows how the system responds when the input is a short pulse, i.e., a Dirac function

$$u(t) = \delta(t)$$

The Laplace transformation is

$$U(s) = \int_0^\infty e^{-st} \delta(t) \mathrm{d}t = 1$$

Hence

$$Y(s) = G(s)U(s) = G(s)$$

Not so common in technological applications, can we think of other applications?

Example - Impulse Response

Let the transfer function of the system be:

$$G(s) = \frac{2}{s^2 + 3s + 2}$$



Shows how the system responds when the input is a step, i.e.,

$$u(t) = \begin{cases} 1 & t \ge 0 \\ 0 & t < 0 \end{cases}$$

The Laplace transformation is

$$U(s) = \int_0^\infty e^{-st} u(t) \mathrm{d}t = \int_0^\infty e^{-st} \mathrm{d}t = -\frac{1}{s} \left[e^{-st} \right]_0^\infty = \frac{1}{s}$$

Very common in technological applications

Example - Step Response

Let the transfer function of the system be:

$$G(s) = \frac{2}{s^2 + 3s + 2}$$



Step Response Analysis

From the last lecture, we know that if the input u(t) is a **step**, then the output in the Laplace domain is

$$Y(s) = G(s)U(s) = G(s)\frac{1}{s}$$

It is possible to do an inverse transform of Y(s) to get y(t), but is it possible to claim things about y(t) by only studying Y(s)?

We will study **how the poles affects the step response**. (The zeros will be discussed later).

Let F(s) be the Laplace transformation of f(t), i.e., $F(s) = \mathcal{L}(f(t))(s)$. Given that the limits below exist¹, it holds that:

Initial value theorem $\lim_{t\to 0} f(t) = \lim_{s\to +\infty} sF(s)$

Final value theorem $\lim_{t\to+\infty} f(t) = \lim_{s\to 0} sF(s)$

For a step response we have that:

$$\lim_{t \to +\infty} y(t) = \lim_{s \to 0} sY(s) = \lim_{s \to 0} sG(s)\frac{1}{s} = G(0)$$

¹Q: When can we NOT apply the Final value theorem?

Some useful matlab commands

- >> s=tf('s'); % enables to use s as transfer fcn
- >> z=0.2; w0=5;
- >> G= w0^2 / (s^2 + 2*z*w0*s + w0^2)
- >> step(G)
- >>
- >> pzmap(G) % pole-zero map





$$G(s) = \frac{K}{1+sT}$$

One pole in s = -1/T

Step response:

$$Y(s) = G(s)\frac{1}{s} = \frac{K}{s(1+sT)} \quad \xrightarrow{\mathcal{L}^{-1}} \quad y(t) = K\left(1-e^{-t/T}\right), \, \mathbf{t} \ge \mathbf{0}$$



Final value:

$$\lim_{t \to +\infty} y(t) = \lim_{s \to 0} sY(s) = \lim_{s \to 0} s \cdot \frac{K}{s(1+sT)} = K$$



T is called the time-constant:

$$y(T) = K(1 - e^{-T/T}) = K(1 - e^{-1}) \approx 0.63K$$

i.e., ${\cal T}$ is the time it takes for the step response to reach 63% of its final value



$$G(s) = \frac{\kappa}{1+sT}$$

Derivative at zero:

$$\lim_{t \to 0} \dot{y}(t) = \lim_{s \to +\infty} s \cdot sY(s) = \lim_{s \to +\infty} \frac{s^2 K}{s(1+sT)} = \frac{K}{T}$$

Second Order System With Real Poles



$$G(s)=rac{K}{(1+sT_1)(1+sT_2)}$$

Poles in $s = -1/T_1$ and $s = -1/T_2$. Step response:

$$y(t) = \begin{cases} \mathcal{K}\left(1 - \frac{T_1 e^{-t/T_1} - T_2 e^{-t/T_2}}{T_1 - T_2}\right), \, \mathbf{t} \ge \mathbf{0} & T_1 \neq T_2 \\ \mathcal{K}\left(1 - e^{-t/T} - \frac{t}{T} e^{-t/T}\right), \, \mathbf{t} \ge \mathbf{0} & T_1 = T_2 = T \end{cases}$$

Second Order System With Real Poles



Final value:

$$\lim_{t \to +\infty} = \lim_{s \to 0} sY(s) = \lim_{s \to 0} \frac{sK}{s(1 + sT_1)(1 + sT_2)} = K$$

Second Order System With Real Poles



$$G(s) = \frac{\kappa}{(1+sT_1)(1+sT_2)}$$

Derivative at zero:

$$\lim_{t\to 0} \dot{y}(t) = \lim_{s\to +\infty} s \cdot sY(s) = \lim_{s\to +\infty} \frac{s^2 K}{s(1+sT_1)(1+sT_2)} = 0$$

$$G(s)=rac{{\cal K}\omega_0^2}{s^2+2\zeta\omega_0s+\omega_0^2}, \quad 0<\zeta<1$$

Relative damping $\zeta,$ related to the angle φ

 $\zeta = \cos(\varphi)$



$$G(s)=rac{{\cal K}\omega_0^2}{s^2+2\zeta\omega_0s+\omega_0^2}, \hspace{1em} 0<\zeta<1$$

Inverse transformation for step response yields:

$$y(t) = K\left(1 - \frac{1}{\sqrt{1 - \zeta^2}}e^{-\zeta\omega_0 t}\sin\left(\omega_0\sqrt{1 - \zeta^2}t + \arccos\zeta\right)\right)$$
$$= K\left(1 - \frac{1}{\sqrt{1 - \zeta^2}}e^{-\zeta\omega_0 t}\sin\left(\omega_0\sqrt{1 - \zeta^2}t + \arcsin(\sqrt{1 - \zeta^2})\right)\right), \mathbf{t} \ge \mathbf{0}$$

$$G(s)=rac{{\cal K}\omega_0^2}{s^2+2\zeta\omega_0s+\omega_0^2}, \hspace{1em} 0<\zeta<1$$

Inverse transformation for step response yields:

$$\begin{aligned} y(t) &= \mathcal{K}\left(1 - \frac{1}{\sqrt{1 - \zeta^2}} e^{-\zeta \omega_0 t} \sin\left(\omega_0 \sqrt{1 - \zeta^2} t + \arccos\zeta\right)\right) \\ &= \mathcal{K}\left(1 - \frac{1}{\sqrt{1 - \zeta^2}} e^{-\zeta \omega_0 t} \sin\left(\omega_0 \sqrt{1 - \zeta^2} t + \arcsin(\sqrt{1 - \zeta^2})\right)\right), \mathbf{t} \ge \mathbf{0} \end{aligned}$$

Exercise: Check of correct starting point of step response.

Step Response

t

$$\begin{aligned} \kappa(0) &= \kappa \left(1 - \frac{1}{\sqrt{1 - \zeta^2}} e^0 \sin \left(\omega_0 \sqrt{1 - \zeta^2} 0 + \arcsin(\sqrt{1 - \zeta^2}) \right) \right) & 1.5 \\ &= \kappa \left(1 - \frac{1}{\sqrt{1 - \zeta^2}} \cdot \sqrt{1 - \zeta^2} \right) \\ &= 0 \\ &= 0 \end{aligned}$$

15

$$G(s) = rac{K\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2}, \quad 0 < \zeta < 1$$

Changing fq $\omega_{\rm 0}$



$$G(s) = rac{K\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2}, \hspace{1em} 0 < \zeta < 1$$

Changing damping ζ



Content

This lecture

- 1. Transient Response
- 2. Step Response Analysis

Next lecture

• Frequency Analysis